Experimental investigation of interaction effects in foam-filled thin-walled aluminum tubes

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Received: 5 August 2004/Accepted: 7 July 2005/Published online: 19 August 2006 © Springer Science+Business Media, LLC 2006

Abstract The interaction coefficients of polystyrene foam filling of thin-walled aluminum cylindrical tubes were investigated experimentally through compression testing of partially foam-filled tubes with and without adhesive. The experimental load-displacement curves and observation of the crushed sections of filled tubes have shown that partial foam filling reduced the fold length and hence increased the average crushing loads of tubes, proving the interaction effect between tube wall and filler. The interaction coefficients for the partial foam filling were further calculated to be in the level and/or higher than that of the foam plateau load of transverse direction.

Introduction

The filling of tubular structures with light-weight foam cores in order for increasing specific energy absorption (SEA) has taken considerable scientific interest. The earliest investigations on the crushing behavior of foam-filled thin-walled sections were conducted by Thornton [1], Lampinen and Jeryan [2], Reid et al. [3] and Reddy and Wall [4]. Recent studies on aluminum (Al) foam-filled thin-walled aluminum and steel tubes

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Center for Materials Research, Izmir Institute of Technology, Gülbahçe Köyü, Urla, Izmir 35430, Turkey e-mail: mustafaguden@iyte.edu.tr the major cause of the reduced fold length in filled tubes [7]. The entrance of the column wall into foam filler results in shorter fold lengths and increased number of folds formed [7]. Santosa et al. [5] pointed out that the encroachment of the column wall into the Al foam filler allows an additional compression in the foam and retards the sectional collapse of the column. Foam filling was also shown to increase the tendency for axisymmetric (concertina) mode of deformation [3]. The shift in the collapse mode from diamond into concertina with foam filling was observed in Al foamfilled Al and steel tubes, polyurethane foam-filled Al tubes and wood sawdust-filled plastic tubes [5, 6, 10, 11] and proposed to be due to the thickening effect of foam filling. It was further shown by Hanssen et al. [8] that after a critical Al foam density the deformation mode in filled Al tubes shifted from diamond to concertina mode. A similar mode-shift was also found in polyurethane foam-filled thin-walled Al tubes with the increasing foam density [10]. Santosa and Wierzbicki [7] showed that the effect of filler on the tube crushing load was similar when the strong axis of the honeycomb through and normal to the compression axis, proving that both axial and lateral strength of the filler were effective in increasing the crushing load of the

tube. Further, Santosa et al. [5] noted that the bonding

between filler and tube wall increased the average

crushing load of filled tube over the unbounded filled

include those of Santosa et al. [5], Seitzberger et al. [6],

Santosa and Wierzbicki [7], Hanssen et al. [8] and

Toksoy et al. [9]. Foam filling generally increases the

average crushing load of filled tube over the sum of the

crushing loads of foam (alone) and tube (alone), which

is known as interaction effect. The restraining effect of

foam filler to the tube folding was also explained to be



tube when appropriate tube geometry and foam density were chosen.

Hannsen et al. [8, 12] developed an equation for the average crushing load of foam-filled ($P_{\rm af}$) columns by including contributions of the average crushing load of empty tube ($P_{\rm ae}$), foam plateau stress ($\sigma_{\rm pl}$) and interaction effect. The equation was found to be well agreed with experimental results and is given as

$$P_{\rm af} = P_{\rm ae} + \sigma_{\rm pl}b^2 + C_{\rm avg}\sqrt{\sigma_{\rm pl}\sigma_{\rm v}}bt \tag{1}$$

where $C_{\rm avg}$, $\sigma_{\rm y}$, b and t are the dimensionless constant which is directly related to the interaction effect, yield strength of the tube material and tube width and thickness, respectively. The second term of the right hand side of the Eq. 1 accounts for the axial compression of the foam and the last term for the interaction effect. If the axial compression of filler is avoided experimentally, the interaction effect between foam filler and tube wall can be determined according to Eq. 1. In this case, the interaction load $(P_{\rm int})$ and interaction coefficient (I) will be

$$P_{\rm int} = P_{\rm fp} - P_{\rm ae} \tag{2}$$

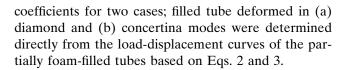
$$I = \frac{P_{\text{int}}}{P_{\text{foam}}} \tag{3}$$

where, $P_{\rm fp}$ and $P_{\rm foam}$ are the average crushing load of foam-filled tube without the axial compression of foam filler and the foam compression load corresponding to the plateau stress, respectively. Santosa et al. [5], based on FEM results, proposed the following equation for the average crushing load of foam-filled square tubes of length b,

$$P_{\rm af} = P_{\rm ae} + C\sigma_{\rm pl}b^2 \tag{4}$$

The constant *C* in Eq. 4 is considered to be the *strengthening coefficient* of foam filling. The values of *C* were shown to be 1.8 and 2.8 for the unbounded and bounded cases, respectively.

The interaction and strengthening coefficients of foam-filled tubes are usually determined by fitting the experimental results of the average crushing loads of empty and foam-filled tubes to Eqs. 1 and 4, respectively. In this study, a novel experimental method was investigated to assess experimentally the interaction effect in thin-walled Al circular tubes deforming in diamond mode (progressive asymmetric). Al tubes in two different tube wall thicknesses and diameters were filled partially with foam (only half of the tube is filled) so that the axial deformation of foam filler was avoided during compression testing. The interaction



Materials and testing

The filler material was a commercially available extruded closed-cell polystyrene foam sheet produced by the Izocam Company of Turkey with a mean density of. 32.1±2 kg m⁻³ determined by dividing the mass of 5 cm long cubic foam sample by its volume. The cells of foam are preferentially elongated through the thickness or rise direction of as-received foam sheets. This gives rise to anisotropy in compression behavior between normal to rise (R), width (W) and extrusion (E)-directions (Fig. 1). The plateau stresses of foams normal to the R, W and E-directions were determined by the compression testing of 5 cm cubic foam samples.

Al (99.7%) drawn tubes, 16 mm (0.22 mm thick) and 25 mm (0.29 mm thick) in outer diameters, were provided by the Metalum Company of Turkey. The average yield and ultimate tensile strength values of tube material were determined by the tensile testing of non-standard small size samples cut axially through drawing direction of the tubes and were found to be 112 and 170 MPa, respectively.

Three groups of compression tests were applied. The first and second groups of tests were on the empty and foam-filled tubes with and without adhesive (Fig. 2a, b). In the last group, partially foam-filled tubes with and without adhesive were tested (Fig. 2c). The foam filling was always through the R direction of foam as shown in Fig. 2b, c. A commercially available foam polystyrene adhesive Bison Styrabond® was used for the bonding of foam-filler to the tube wall. A tube

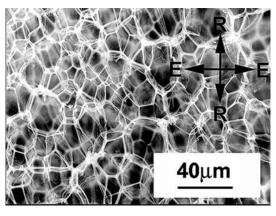


Fig. 1 The cell morphology through the rise and the extrusion direction of the foam



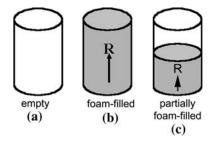


Fig. 2 Schematic of three groups of the compression tests applied to (a) empty, (b) foam-filled and (c) partially foam-filled tubes

length to diameter ratio of 1.6, which avoided tube global buckling, was used in all tests.

Compression tests were conducted using a displacement controlled Schimadzu AG-I universal test machine. Empty and foam-filled tubes were compressed at a cross-head speed of 2.5 mm min $^{-1}$ with a corresponding deformation rate of 1×10^{-3} s $^{-1}$. The average crushing load of empty and filled tubes are calculated using the following relations,

$$P_{\rm ae}(\delta) = \frac{\int_0^\delta P \, \mathrm{d}\delta}{\delta} \tag{5}$$

where P and δ are the load and displacement, respectively. The weights of tubes were measured before and after filling, hence the weight of adhesive used was calculated. The amount of adhesive used varied between 3 and 6% of the total weight of filled tube.

Results and discussions

Figure 3 shows the compression nominal stress–strain curves of the foam tested normal to the R, W and E-directions. Anisotropy in compression behavior is clearly seen in this figure; foam shows a higher plateau stress in the R-direction. As shown in Fig. 3, foam stress in the plateau region is nearly constant until about 0.4 and 0.2 strains for R and W-directions, respectively.

Empty tubes deformed in diamond mode with 6 and 8 corners per fold and totally 9–10 and 7–8 folds formed in 16 and 25 mm empty tubes, respectively. It is also noted that folding in both tubes started in axisymmetric mode before reverting into diamond mode, a phenomenon observed in multi-mode collapse mode [13].

Similar to empty tubes, foam-filled 16 mm Al tube collapsed in diamond mode and totally 10–12 folds formed. In partially foam-filled 16 mm Al tube with and without adhesive the deformation mode remained

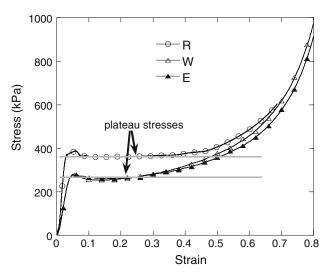


Fig. 3 Nominal compression stress-strain curves of the foam through R, E and W directions

to be the same with that of empty tube (diamond), but folding started either at the filled (sample A) or non-filled end (sample B) of the tube (Fig. 4a, b). The load-displacement and average crushing load -displacement curves of foam-filled, partially foam-filled sample A and empty tubes are shown in Fig. 5(a, b), respectively. The partial foam filling of samples A increases the load and average crushing load values and decreases the fold length (Fig. 5a, b). It is also noted that the load-displacement pattern and fold length of partially foam-filled tube are very similar to those of foam-filled tube except the load values are lower in partially filled tube (Fig. 5a). The average crushing load of partially foam-filled tube is slightly higher than that of foam (R)+empty tube (foam alone + tube alone) while

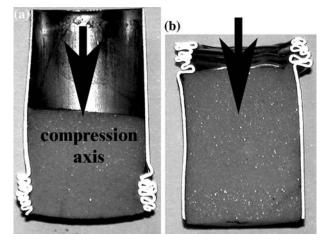


Fig. 4 Compressed sections of partially foam-filled 16 mm tube, folding starts (a) at the filled end (sample A) and (b) at the non-filled end (sample B)



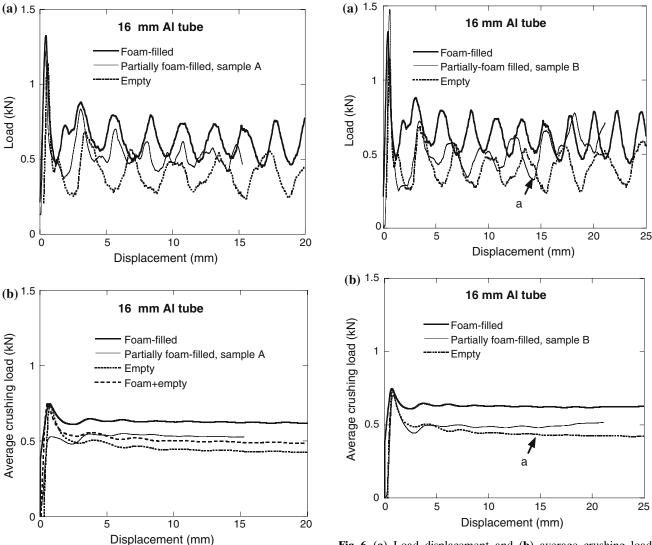


Fig. 5 (a) Load displacement and (b) average crushing load-displacement curves of empty, partially filled (sample A) and filled 16 mm tube with adhesive

Fig. 6 (a) Load displacement and (b) average crushing load-displacement curves of empty, partially filled (sample B) and filled 16 mm tube with adhesive

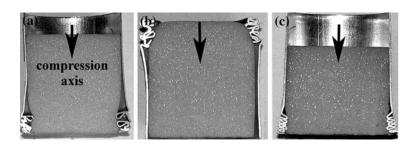
lower than that of filled tube. In samples B of partially foam-filled 16 mm tubes, the fold length and the load values are more or less similar to those of empty tube until the point a of Fig. 6(a), at which folding starts to proceed in the foam-filled section, thereafter, due to the axial compression of the filler the load values increase and the fold length decreases to the level of filled tube. The average crushing load of partially foam-filled sample B is also noted to be slightly higher than that of empty tube (Fig. 6b). In partially foam-filled 16 mm tubes without adhesive the load-displacement curves are also found to be very similar to those of filled tubes with adhesive.

In the partial filled 25 mm tubes with and without adhesive, similar to 16 mm tubes, folding started either at the filled (sample A) or at the non-filled

(sample B) end (Fig. 7a, b). The use of adhesive in samples A however shifted the deformation mode from diamond into concertina (Fig. 7c) while empty and partailly foam-filled without adhesive tubes deformed into diamond mode. The load-displacement and average crushing load-displacement curves of 25 mm Al empty and foam-filled and partially foamfilled (sample A) tubes without adhesive are shown in Fig. 8(a, b) respectively. The effect of foam filling in samples A without adhesive, is the reduction of the fold length and increase of the average crushing load values over the empty tubes to a level of foam (R)+empty tube (Fig. 8b). The load-displacement characteristic and fold length of samples A of partially foam-filled tube with adhesive are very similar to that of foam-filled tube except the average load



Fig. 7 Crushed cross-sections of partially foam-filled 25 mm tube, folding starts (a) at the filled end (sample A) and (b) at the non-filled end (sample B) of without adhesive and (c) sample A with adhesive



values are lower in partially filled tube (Fig. 9a, b). The average crushing load of partially foam-filled tube in these samples is higher than that of foam (R)+empty tube (Fig. 9b). In patially foam-filled

25 mm tube of samples B, the fold length and load values are however the same with those of empty tubes until the point a of Fig. 10, at which folding starts to proceed in the filled section. Thereafter, load

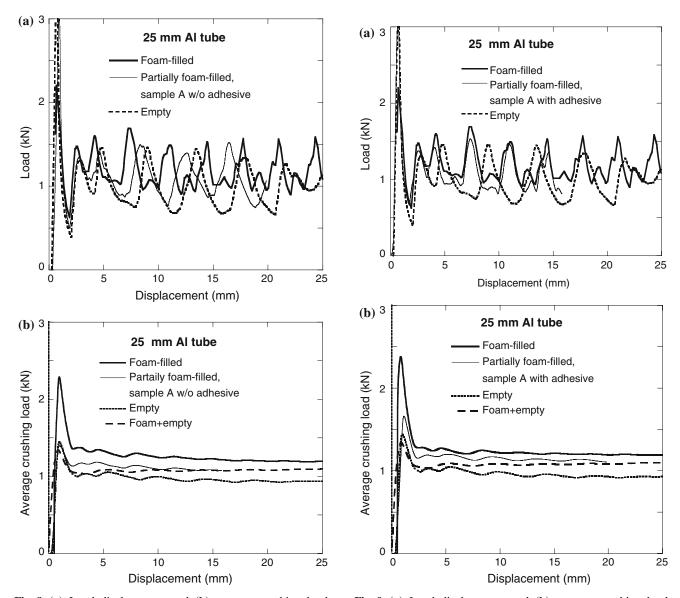


Fig. 8 (a) Load displacement and (b) average crushing load-displacement curves of empty, partially filled (sample A) and filled 25 mm tube without adhesive

Fig. 9 (a) Load displacement and (b) average crushing load-displacement curves of empty, partially filled (sample A) and filled 25 mm tube with adhesive



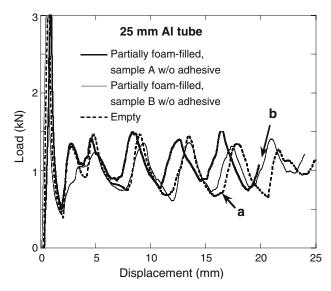


Fig. 10 Load displacement curves partially foam-filled sample with and without adhesive (sample B) and empty tubes of 25 mm tube

values increase to those of sample A without adhesive (point b) in Fig. 10.

It was noted the triggering position of folding in partially filled tubes was random and might be affected by the geometrical parameters of foam-filled tube and the constraints imposed by the compression test plates. Tests were conducted until about at least 3 consistent load-displacement curves obtained for each configuration and sample type. Figure 11 shows the average crushing loads of filled, partially foam-filled and empty tubes as well as the variations in the average crushing loads between 3 tests conducted at the same configuration. For comparison, the load-displacement curves

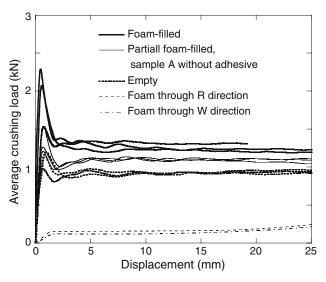


Fig. 11 Average crushing loads of filled, partially foam-filled and empty 25 mm tubes



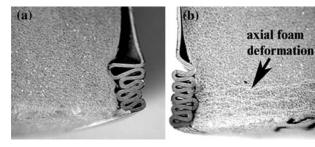


Fig. 12 Crushed cross-sections of partially foam-filled 25 mm tubes (a) without adhesive and (b) with adhesive

of the foam in R and W directions are also shown in the same figure.

The important deformation parameters of empty, partially foam-filled and foam-filled 16 and 25 mm Al tubes are tabulated in Tables 1 and 2, respectively. In these tables, ΔP_a refers to the increase in average crushing load of tubes with foam filling. The relatively high values of initial peak loads were effective in increasing average crushing loads, particularly at low displacements. The average crushing loads were also calculated by simply changing the integration limits in Eq. 5 and found that the difference between the average crushing loads with and without excluding initial peak loads diminished with increasing displacement and became negligible (3% difference) at 15 mm displacement. The average crushing load values were therefore calculated at 15 mm displacement. The effect of partial foam filling on the crushing behavior of tested Al tubes can be also noted in Tables 1 and 2; the fold lengths reduced and average crushing loads increased. It is also noted in Table 1 that the fold lengths of partially foam-filled and foam-filled 16 mm Al tubes are very similar. A similar behavior is also found in partially foam-filled with adhesive and foam-filled 25 mm Al tubes (Table 2). The interaction coefficients listed in Tables 1 and 2 were calculated based on the foam plateau load through R and transverse directions. The first numbers in these tables show the calculations based on through R direction foam plateau load and the second based on the transverse to R-direction plateau load. The calculations based on foam plateau load in the R-direction give the interaction coefficients of 1.47 for 16 mm partially filled tubes and 0.96 and 1.14 for 25 mm partially filled tubes without and with adhesive, respectively. Based on foam transverse plateau load, the interaction coefficients for 16 mm partially filled is 1.7, for 25 mm partially filled with adhesive is 1.3 and for 25 mm partially filled with adhesive is 1.1, about unity.

The increase in the average crushing loads of partially foam-filled tube (without adhesive) with respect

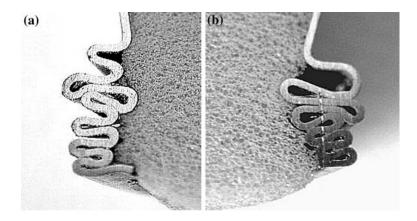
Table 1 Crushing parameters of empty, partially foam-filled and foam-filled 16 mm Al tubes (average of at least 3 tests)

Material	$P_{\rm a}$ (kN)	$\Delta P_{\rm a} (P_{\rm af} - P_{\rm ae})$ (kN)	Number of folds	Fold Length (mm)	Interaction Coefficient (I)	Def. Mode
Empty	0.43	0	9–10	3.4–3.8		diamond
Partially filled (adhesive)	0.53	0.1		2.2-2.3	1.4–1.7	diamond
Filled	0.63	0.2	10–11	2.2–2.5		diamond

Table 2 Crushing parameters of empty, partially foam-filled and foam-filled 25 mm Al tubes (average of a least 3 tests)

Material	$P_{\rm a}$ (kN)	$\Delta P_{\rm a} (P_{\rm af} - P_{\rm ae})$ (kN)	Number of folds	Fold Length (mm)	Interaction Coefficient (I)	Def. Mode
Empty Partially filled Partially filled (adhesive) Filled	0.94 1.1 1.13 1.23	0 0.16 0.19 0.29	7–8 9–10	4.1–4.5 3.9–4 3.2–3.5 3.2–3.5	0.96–1.1 1.14–1.3	diamond diamond concertina concertina

Fig. 13 Crushed crosssections of partially foamfilled 16 mm tubes (a) without adhesive and (b) with adhesive



to the empty tube is mainly due to the resistance of foam filler to the inward folding of the thin-walled tube, leading to shorter fold lengths and hence increased crushing loads. The interaction effect, which was previously shown to be in the level of the foam strength for the square tubes [7], was in accord with the partially filled 25 mm without adhesive. In 16 mm and 25 mm with adhesive Al tubes the interaction coefficient is however higher than the foam transverse plateau load (1.7 and 1.3). A close inspection of the crosssection of the partially deformed 25 mm without adhesive tubes showed only lateral contraction of the filler (Fig 12a) while in 25 mm with adhesive tube besides lateral contraction the axial deformation of the filler was observed (Fig 12b). A similar effect, deformation in the axial direction of filler was also observed in 16 mm partially filled with and without adhesive tubes as shown in Fig. 13(a, b). These results tended to confirm an interaction coefficient higher than filler plateau load in the case of axial deformation of the filler. One of the reasons for the observed axial deformation of the filler might be the filler entrance in

between the folds, leading to a higher strengthening than the transverse filler load.

In the filled tubes, the interaction effect may be partly due to the resistance of filler to the inward and/ or outward folding of tube wall and partly due to the interfacial friction stress between foam and tube wall. Numerical studies of Al foam-filled tubes have shown a negligible effect of interfacial frictional stress on the crushing strength of tubes [5]. The use of adhesive can contribute to the specific energy absorption of tube by two mechanisms, namely, increased load transfer from tube wall to the foam core and peeling of the adhesive. The latter mechanism occurs mainly due to the outward folding of the tube. In the partially filled 25 mm Al tube with adhesive, the filler remained to be attached to the tube wall throughout the crushing process. This is likely to result in higher load transfer to the foam filler and change of deformation mode.

The present results on the partially foam-filled thinwalled Al tubes have shown few effects of the filler on the crushing behavior of thin walled tubes. In the absence of axial compression of foam filler, the resis-



tance to tube folding provided by the filler is about in the level of filler lateral load. The resistance of foam filler in partially filled tubes reduces the fold length in the level of the filled tubes. However, the results should be taken cautiously since the axial deformation of the filler may results in changes in the lateral resistance of filler to the tube folding in filled tubes.

Conclusions

The interaction coefficients of polystyrene foam filling of thin-walled aluminum cylindrical tubes, 16 mm and 25 mm in diameter, were investigated experimentally through compression testing of partially foam-filled tubes with and without adhesive. In partially foam-filled 16 mm tube the deformation mode was the same with those of empty and filled tube. In partially filled 25 mm tubes without adhesive the deformation mode was diamond while the use of adhesive reverted the deformation mode in the partially filled tube into concertina, which the same with that of filled tube. The effect of partial foam filling was to reduce the fold length and increase the average crushing loads of tubes. The interaction coefficients for the partial foam

filling were further shown to be in the level and/or higher than that of the foam plateau loads.

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